JC20 Rec'd PCT/PTO OCT 0 1 200% ATTORNEY'S DOCKET NUMBER (1390 REV. 5-93) US DEPT. OF COMMERCE PATENT & TRADEMARK OFFICE 110748 TRANSMITTAL LETTER TO THE U.S. APPLICATION NO. **UNITED STATES** (if known, sec 37 C.F.R.1.5) DESIGNATED/ELECTED OFFICE (DO/EO/US) CONCERNING A FILING 09/937776 UNDER 35 U.S.C. 371 AFRON NO. INTERNATIONAL APPLIA INTERNATIONAL FILING DATE PRIORITY DATE CLAIMED PCT/NL00/00227 April 6, 2000 April 6, 1999 OCT 0 1 2001 TITLE OF INVENTION ACTUATOR HAVING COMPACT GEAR EDUCTION APPLICANTS FOR DO/EO Hendrikus Jan KAPAAN, Jacobas Zw WARTS, Simon Jan BROERSEN Applicant herewith submits to the United States Designated/Elected Office (DO/EO/US) the following items and other information: This is a **FIRST** submission of items concerning a filing under 35 U.S.C. 371. 1. 2. This is a **SECOND** or **SUBSEQUENT** submission of items concerning a filing under 35 U.S.C. 371. 3. This express request to begin national examination procedures (35 U.S.C. 371(f)) at any time rather than delay examination until the expiration of the applicable time limit set in 35 U.S.C. 371(b) and PCT Articles 22 and 39(1). A proper Demand for International Preliminary Examination was made by the 19th month from the earliest claimed priority date. l, i, j 5 A copy of the International Application as filed (35 U.S.C. 371(c)(2)) a. is transmitted herewith (required only if not transmitted by the International Bureau). b. A has been transmitted by the International Bureau. c. is not required, as the application was filed in the United States Receiving Office (RO/US) 6. A translation of the International Application into English (35 U.S.C. 371(c)(2)). 7. Amendments to the claims of the International Application under PCT Article 19 (35 U.S.C. 371(c)(3)) a. \_\_\_ are transmitted herewith (required only if not transmitted by the International Bureau). have been transmitted by the International Bureau. c. In have not been made; however, the time limit for making such amendments has NOT expired. : #5 d. have not been made and will not be made. A translation of the amendments to the claims under PCT Article 19 (35 U.S.C. 371(c)(3)). 8. 9. An oath or declaration of the inventor(s) (35 U.S.C. 371(c)(4)). 10. A translation of the annexes to the International Preliminary Examination Report under PCT Article 36 (35 U.S.C. 371 (c)(5)). Items 11. to 16. below concern other document(s) or information included: 11. An Information Disclosure Statement under 37 CFR 1,97 and 1,98. 12. An assignment document for recording. A separate cover sheet in compliance with 37 CFR 3.28 and 3.31 is included.  $\boxtimes$ 13. A FIRST preliminary amendment. A SECOND or SUBSEQUENT preliminary amendment. 14. A substitute specification. 15. Entitlement to small entity status is hereby asserted.

Other items or information: A submission of the annexes to the International Preliminary Examination

Report under PCT Article 36 (35 U.S.C. 371 (c)(5)).

16. 🖂

JC05 Rec'd PCT/PTO 0 1 OCT 2001

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17. X The following	7.  The following fees are submitted:		CALCU	JLATIONS	PTO USE ONLY	
Basic National fee (37 CFR 1.492(a)(1)-(5)):						
Search Report has been prepared by the EPO or JPO\$890.00						
International preliminary examination fee paid to USPTO (37 CFR1.482)\$710.00						
No international preliminary examination fee paid to USPTO (37 CFR 1.482) but international search fee paid to USPTO (37 CFR 1.445(a)(2))\$740.00						
Neither international preliminary examination fee (37 CFR 1.482) nor international search fee (37 CFR 1.445(a)(2)) paid to USPTO\$1,040.00						
International preliminary examination fee paid to USPTO (37 CFR 1.482) and all claims satisfied provisions of PCT Article 33(2)-(4)\$ 100.00						
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Total Claims	15- 20 =	0	X \$ 18.00	\$		
Independent Claims	1- 3 =	0	X \$84.00	\$		
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SUBTOTAL =			\$890.00			
Processing fee of \$130.00 for furnishing the English translation later than ☐ 20 ☐ 30 month from the earliest claimed priority date (37 CFR 1.492(f)).			\$			
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					Amount to be refunded	\$
					Charged	\$
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to Deposit Account No. <u>15-0461</u> . A duplicate copy of this sheet is enclosed.  NOTE: Where an appropriate time limit under 37 CFR 1.494 or 1.495 has not been met, a petition to revive (37 CFR						
1.137(a) or (b)) must be filed and granted to restore the application to pending status.						
SEND ALL CORRESPONDENCE TO: OLIFF & BERRIDGE, PLC						
P.O. Box 19928 Alexandria, Virginia 22320 NA				NAMI Jame REGISTRATIO	S A. ONF ON NUMBER: 2	-  27,075
				NAME: Joel S REGISTRATIO	6. Armstrong	36 <b>4</b> 30



### PATENT APPLICATION

### IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

In re the Application of

Hendrikus Jan KAPAAN, Jacobus ZWARTS, Simon Jan BROERSEN

Application No.: U.S. National Stage of PCT/NL00/00227

Filed: October 1, 2001 Docket No.: 110748

For: ACTUATOR HAVING COMPACT GEAR REDUCTION

### PRELIMINARY AMENDMENT

Director of the U.S. Patent and Trademark Office Washington, D. C. 20231

Sir:

Prior to initial examination, please amend the above-identified application as follows:

### IN THE CLAIMS:

Please replace claims 6-15 as follows:

- 6. (Amended) Actuator according to claim 1, wherein the extenric gear wheel (24) is rotatably supported with respect to the excentric hub (23) by means of a rolling element bearing (30).
- 7. (Amended) Actuator according to claim 1, wherein the motor (2) is an electric motor, the stator (6) of which is connected to the housing (1).
- 8. (Amended) Actuator according to claim 1, wherein the gear reduction mechanisme (4) is at the end of the screw mechanism (3) opposite the end thereof engaging an actuating means (38) for a brake pad (39).
- 9. (Amended) Actuator according to claim 1, wherein the screw (16) of the screw mechanism (3) is rotatably supported by means of a support bearing (11) with respect to a

central support shaft (13), the gear ring (23) and the gear wheel (24) of the reduction gear mechanism (4) surrounding said central support shaft (13).

- 10. (Amended) Actuator according to claim 1, wherein the screw (16) has a bore (35) containing a lubricant reservoir (36).
- 11. (Amended) Actuator according to claim 1, wherein the gear reduction mechanism (4) and a positive back-drive mechanism (37) are contained in a gear reduction module (40).
- 12. (Amended) Actuator according to claim 1, wherein the gear reduction module (4) comprises a central support shaft (13) for supporting the screw mechanism (3).
- 13. (Amended) Actuator according to claim 1, wherein the screw mechanism (3), a support bearing (11) for supporting the screw mechanism (3), the rotor (7) of the motor (2) as well as a bearing (9) for supporting the rotor (7) on the screw mechanism (3) are contained in a actuator module (41).
- 14. (Amended) Actuator according to claim 1, wherein the housing (1), the stator (6) and electric connections for the motor (2) are contained in a housing module (42).
- 15. (Amended) Brake caliper, comprising a claw piece with at least two brakes, and an actuator according to claim 1.

## **REMARKS**

Claims 1-15 are pending. By this Preliminary Amendment, claims 6-15 are amended to eliminate multiple dependencies and typographical errors. Prompt and favorable examination on the merits is respectfully requested.

The attached Appendix includes marked-up copies of each rewritten claim (37 C.F.R. §1.121(c)(1)(ii)).

Respectfully submitted,

James A. Oliff Registration No. 27,075

Joel S. Armstrong Registration No. 36,430

JAO:JSA/cmm

Attachment: Appendix

Date: October 1, 2001

OLIFF & BERRIDGE, PLC P.O. Box 19928 Alexandria, Virginia 22320 Telephone: (703) 836-6400 DEPOSIT ACCOUNT USE
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### **APPENDIX**

Changes to Claims:

The following are marked-up versions of the amended claims:

- 6. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the extensic gear wheel (24) is rotatably supported with respect to the excentric hub (23) by means of a rolling element bearing (30).
- 7. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the motor (2) is an electric motor, the stator (6) of which is connected to the housing (1).
- 8. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the gear reduction mechanisme (4) is at the end of the screw mechanism (3) opposite the end thereof engaging an actuating means (38) for a brake pad (39).
- 9. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the screw (16) of the screw mechanism (3) is rotatably supported by means of a support bearing (11) with respect to a central support shaft (13), the gear ring (23) and the gear wheel (24) of the reduction gear mechanism (4) surrounding said central support shaft (13).
- 10. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the screw (16) has a bore (35) containing a lubricant reservoir (36).
- 11. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the gear reduction mechanism (4) and a positive back-drive mechanism (37) are contained in a gear reduction module (40).
- 12. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the gear reduction module (40) comprises a central support shaft (13) for supporting the screw mechanism (3).
- 13. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the screw mechanism (3), a support bearing (11) for supporting the screw mechanism (3), the

rotor (7) of the motor (2) as well as a bearing (9) for supporting the rotor (7) on the screw mechanism (3) are contained in a actuator module (41).

- 14. (Amended) Actuator according to any of the preceding claims, claim 1, wherein the housing (1), the stator (6) and electric connections for the motor (2) are contained in a housing module (42).
- 15. (Amended) Brake caliper, comprising a claw piece with at least two brakes, and an actuator according to any of the preceding claims. claim 1.

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### Actuator having compact gear reduction

The invention is related to an actuator, comprising a housing which contains a motor and a screw mechanism, said screw mechanism comprising a screw and a nut one of which is rotatably supported with respect to the housing, and a gear reduction mechanism connecting the rotor of the motor to the rotatable screw or nut.

Such actuator is generally known, and can be applied for various purposes such as for actuating a brake, a clutch etcetera. In many of these applications, it is desirable to have a large reduction between the motor and the object to be actuated, such as brake pads. On the other had, overall dimensions and weight should remain limited.

The screw mechanism itself provides a reduction which i.a. depends on the pitch of the threads. A small pitch however entails rather narrow thread, which in particular for ball screws would lead to small, vulnerable balls which moreover are difficult to handle.

The gear reduction means therefore preferably should provide a relatively large reduction, such that the screw mechanism can have coarser threads which provide a lower reduction of said screw mechanism itself.

In this respect, the prior art gear reduction mechanisms are not adequate. They comprises a ring gear, satellite wheels and a crown gear. Usually, two or more of these mechanisms are arranged in series so as to obtain the required reduction. The overall dimensions and weight are high.

The object of the invention is therefore to provide an actuator having a gear reduction means with a large reduction, and with relatively small dimensions and low weight. This object is achieved in that the gear reduction mechanism comprises at least a concentric gear ring with radially inwardly directed teeth, an excentrically positioned gear wheel having radially outwardly directed teeth wherein the outer diameter of the gear wheel is smaller than the inner diameter of the gear ring, such that the teeth of said gear wheel and gear ring engage each other along a part of their circumferences, and at an opposite part of their circumferences are out of engagement, said gear wheel being rotatable accommodated on a extentric hub which is connected to the rotor of the motor.

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The reduction gear means according to the invention provides a large reduction as a result of the extentric position of the gear wheel and hub. Its dimensions are small in thickness direction, leading to a compact embodiment.

Preferably, the gear ring is integrated with the screw of the screw mechanism, said screw being rotatably supported with relation to the housing.

Furthermore, a very compact actuator is obtained in case the rotor of the motor is rotatably supported on the outer ring of a support bearing, said outer ring being integrated with the screw and the gear ring.

By means of a radially inwardly extending flange, the rotor may be connected to the excentric hub.

Also, a positive back-drive mechanism is connected to the flange and the housing.

The actuator may be assembled from several sub assemblies such as a housing module, an actuator module and a gear reduction module.

According to a further embodiment, the housing hay have a bore accommodating at least the nut and/or the screw, and an axially fixed part of said nut or screw, supported with respect to a radial support abutment, which extends inwardly in the bore. In this embodiment, the rotor of the motor is supported rotatably on a sleeve, said sleeve engaging the fixed part and extending away from an actuator head which is movable in axial direction by means of the nut or the screw, said sleeve having a radially outwardly extending sleeve flange which is interposed between said support abutment, and the axially fixed part.

In particular, the flange of the sleeve is supported on an abutment surface of the support abutment, which faces the actuating head connected to the axially displaceable nut or screw for exerting a compressive force.

The rotor sleeve is now held firmly clamped between the radial support abutment and the axially fixed part of the nut or screw, which provides a simple, reliable construction. No additional fastening means are necessary for supporting the rotor sleeve.

In a practical embodiment, the nut is fixedly supported within the housing, said nut having a radially outwardly extending nut flange facing the outwardly extending sleeve flange and overlapping the inwardly extending actuator support abutment.

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In service of the screw actuator according to the invention, misalignment may occur as a result of eccentric forces and/or transverse forces. Such misalignment might cause damage to the balls and raceways of the nut and the screw, which are rather vulnerable to such loadings. According to the invention, this problem can be alleviated in case the outwardly facing surfaces of sleeve flange and the nut flange are curved in axial cross section, so as to allow swivelling or tilting of said nut and sleeve due to misalignment forces.

The screw/nut unit, as well as the drive unit including rotor and possible reduction means, may now move in unison so as to adapt to the misalignment.

A very stable embodiment is obtained in case the nut has a nut extension extending beyond the nut flange and inside the support abutment, the sleeve having an axially extending support part which is accommodated between said nut extension and the support abutment.

According to a preferred embodiment, the actuating head engages the screw through a rolling element bearing. Said rolling element bearing is integrated with the actuating head. A further improvement concerning stability is obtained in case the actuating head is accommodated in a cylinder, which is held non-rotatably in the bore.

The sleeve may comprise a sheet metal part which is provided with the inner reaceway of at least one support bearing for rotatably supporting the rotor; alternatively, a separate bearing may be mounted with its inner ring on said sleeve.

For ease of handling and assembly, the sleeve and the nut are clampingly preassembled.

The sleeve may furthermore comprise an inwardly extending flange at its end opposite the sleeve flange, which flange carries a gear wheel of the gear reduction mechanism.

The invention will be explained further with reference to an embodiment shown in the figures.

Figure 1 shows a longitudinal section of a brake calliper comprising an actuator according to the invention.

Figure 2 shows a cross-section according to II-II of figure 1.

Figure 3 shows a further embodiment.

The actuator according to the invention comprises a housing 1, containing an electric motor 2, a gear reduction mechanism 4 and a screw mechanism 3. The housing

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at its one end comprises an external screwthread 5, by means of which the actuator can be connected to e.g. a brake calliper in the case of a disc brake application.

The stator 6 of the motor 2 is connected to the housing 1, the rotor 7 of the motor is supported on a sleeve 8. This sleeve 8, by means of ball bearings 9, is in turn supported with respect to the outer ring 10 of a bearing 11. The inner ring 12 of said bearing 11 is supported on a central support shaft 13, connected to the housing 1.

The central support shaft 13 engages the housing 1 through a load cell 15, and is snapfitted by means of clipring 43 accommodated in the groove 44 of the inner ring 12.

The outer ring 10 of the bearing 11 is integrated with screw 16 of screw mechanism 3. This screw 16 is thus rotatably supported with respect to the housing 1. The nut 17 of the screw mechanism 4 is slidably, but not rotatably with respect to the housing. Through the screwthreads 18, 19 and balls 20, the rotary motion of the screw 16 is converted into a linear motion of the nut 17, which is contained in a cylinder space 32 in the housing 1.

According to the invention, the rotor 7 and the sleeve 8 carry an inwardly direted flange 21, which carries an excentric hub 23.

The excentric hub 23 rotatably supports a gear wheel 24 through bearing 30, the outer teeth of which gear wheel 24 engage the inwardly directed teeth of the ring gear 25.

The outer diameter of the gear wheel 24 is smaller than the internal diameter of the gear ring 25, such that over certain distance the circumferences of these gear members engage each other. At the opposite circumferential parts, they are not in engagement, which means that the gear 24 is able to excentrically rotate with respect to the gear ring 25.

In this manner, a great reduction of the rotations of the rotor 7 is obtained.

The brake calliper 51, shown is figure 3 comprises a screw actuator having a nut 55, which by means of balls 78 rotatably supports a screw 56. The nut 55 and screw 56 have appropriately shaped scew type grooves 81.

The electric motor 57 has a stator 58 connected to the housing 51, as well as a rotor 59 which through bearings 69 is rotatably supported on a sleeve 63. That sleeve 63 comprises a raceway 68 for the bearings 69. In the alternative, the sleeve may of course carry the inner rings of separate bearings.

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Said rotor sleeve 69 has an outwardly extending sleeve flange, which is held between a radially inwardly extending support abutment 61 of the housing 51, as well as an outwardly extending nut flange 65.

Said outwardly extending nut flange 65, as well as the sleeve flange 63 are sitting within the bore 60 in the housing 51. The support abutment 61 extends radially inwardly with respect to the wall of the bore 60.

Non-rotatably held within the bore 60 is a cylinder 72, within which an actuating head 71 is accommodated. Through ball bearing 73, that actuating head 71 engages the screw 56 of the screw actuator 54.

By actuating the motor 57, the rotor 59 drives the screw 56 of the screw actuator 54, via the shaft 76. Said shaft 76 is connected on the one hand to the excentric reduction gear mechanism 4 according to firgures 1 and 2, which is driven by the rotor 59 through the rotor sleeve 63 and the inwardly extending flange 74 thereof.

On the other hand, central drive shaft 76 extends into a bore 77 of the screw 56, which shaft 76 is non-rotably coupled to the screw 56 through a spline/groove mechanism.

As a result, the screw 45 is rotated and moved outwardly, so as to bring the brake pad 53 closer to brake pad 52 for exerting a braking effect on a brake disk (not shown).

As a result of the compressive forces thereby exerted on the nut 55, the support sleeve 63 is held firmly clamped against the support abutment 61 of the housing 51.

With the aim of accommodating misalignments, which might cause harm to the raceways and balls of the screw actuator, the sleeve flange 64 as well as the nut flange 65 may be curved preferably along the surface of an imaginary sphere, so as to allow some adaptation of the screw actuator 54 together with the rotor 59 and reduction mechanism 70.

The screw 56 can be driven by means of an auxiliary drive, e.g. a hand brake drive 80, connected to drive shaft 76.

### Claims

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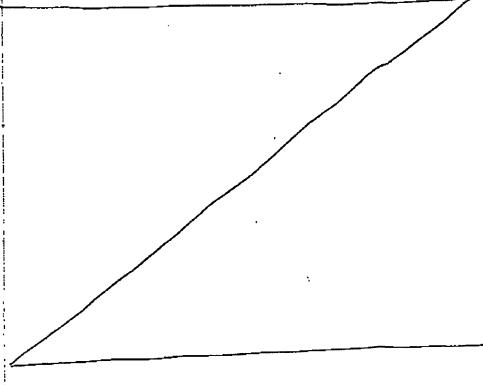
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1. Actuator, comprising a housing (1), which contains a motor (2) and a screw mechanism (3), said screw mechanism (3) comprising a screw (16) and a nut (17) one of which is rotatably supported with respect to the housing (1), and a gear reduction mechanism (4) connecting the rotor (7) of the motor (2) to the rotatable scew (16) or nut (17), said gear reduction mechanism (4) comprising at least a concentric gear ring (25) with radially inwardly directed teeth, an excentrically positioned gear wheel (24) having radially outwardly directed teeth wherein the outer diameter of the gear wheel (24) is smaller than the inner diameter of the gear ring (25), such that the teeth of said gear wheel (24) and gear ring (25) engage each other along a part of threir circumferences, and at an opposite part of their circumferences are out of engagement, said excentric gear wheel (24) being rotatable accommodated on an excentric hub (23) which is connected to the rotor (7) of the motor (2), characterised in that the gear ring (25) is integrated with the screw (16) of the screw mechanism (3), said screw (16) being rotatably supported with relation to the housing (1).



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### <u>Claims</u>

Actuator, comprising a housing (1), which contains a motor (2) and a screy mechanism (3), said screw mechanism (3) comprising a screw (16) and a nut (17) one of which is rotatably supported with respect to the housing (1), and a grear reduction mechanism (4) connecting the rotor (7) of the motor (2) to the rotatable scew (16) or nut (17), characterized in that the gear reduction mechanism (4) comprises at least a concentric gear ring (25) with radially inwardly directed teeth, an excentrically positioned gear wheel (24) having radially outwardly directed teeth wherein the outer diarneter of the gear wheel (24) is smaller than the inner diameter of the gear ring (25), such that the teeth of said gear wheel (24) and gear ring (25) engage each other along a part of their circumferences, and at an opposite part of their circumferences are out of engagement, said excentric gear wheel (24) being rotatable accommodated on an excentric hub (23) which is connected to the rotor (7) of the motor (2).

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2. Actuator according to claim 1, wherein the gear ring (25) is integrated with (16) of the screw mechanism (3), said screw (16) being rotatably supported with relation to the housing (1).

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Actuator according to claim 1 42, wherein the rotor (7) of the motor (2) is rotaliably supported on the outer ring (10) of a support bearing (11), said outer ring (10) being integrated with the screw (16) and the gear ring (25).

Actuator according to claim 7, wherein the rotor (7) by means of a radially inwardly extending flange (21) is connected to the excentric hub (23).

Actuator according to claim 4, wherein a positive back-drive mechanisme (30) is connected to the flange (21) and the housing (1).

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Actuator according to claim s, wherein the positive back-drive mechanism is a spiral spring (30).

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Actuator according to any of the preceding claims, wherein the extenric gear

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wheel (24) is rotatably supported with respect to the excentric hub (23) by means of a rolling element bearing (30).

- Actuator according to any of the preceding claims, wherein the motor (2) is an electric motor, the stator (6) of which is connected to the housing (1).
- Actuator according to any of the preceding claims, wherein the g-ear reduction mechanisme (4) is at the end of the screw mechanism (3) opposite the end thereof engaging an actuating means (38) for a brake pad (39).
- Actuator according to any of the preceding claims, wherein the screw (16) of the screw mechanism (3) is rotatably supported by means of a support bearing (11) with respect to a central support shaft (13), the gear ring (23) and the gear wheel (24) of the reduction gear mechanism (4) surrounding said central support shaft (13).
  - Actuator according to any of the preceding claims, wherein the screw (16) has a bore (35) containing a lubricant reservoir (36).
- 12. Actuator according to any of the preceding claims, wherein the gear reduction mechanism (4) and a positive back-drive mechanism (37) are contained in a gear reduction module (40).
  - Actuator according to any of the preceding claims, wherein the gear reduction module (40) comprises a central support shaft (13) for supporting the screw mechanism (3).
- Actuator according to any of the preceding claims, wherein the screw mechanism (3), a support bearing (11) for supporting the screw mechanism (3), the rotor (7) of the motor (2) as well as a bearing (9) for supporting the rotor (7) on the screw mechanism (3) are contained in a actuator module (41).
  - 13. Actuator according to any of the preceding claims, wherein the housing (1),

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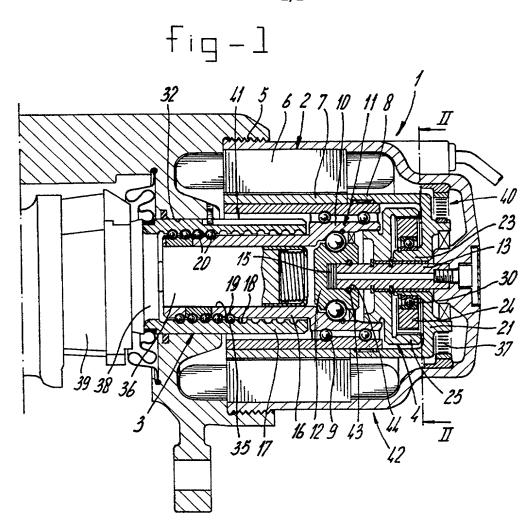
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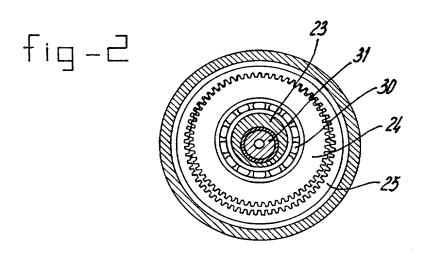
the stator (6) and electric connections for the motor (2) are contained ir a housing module (42).

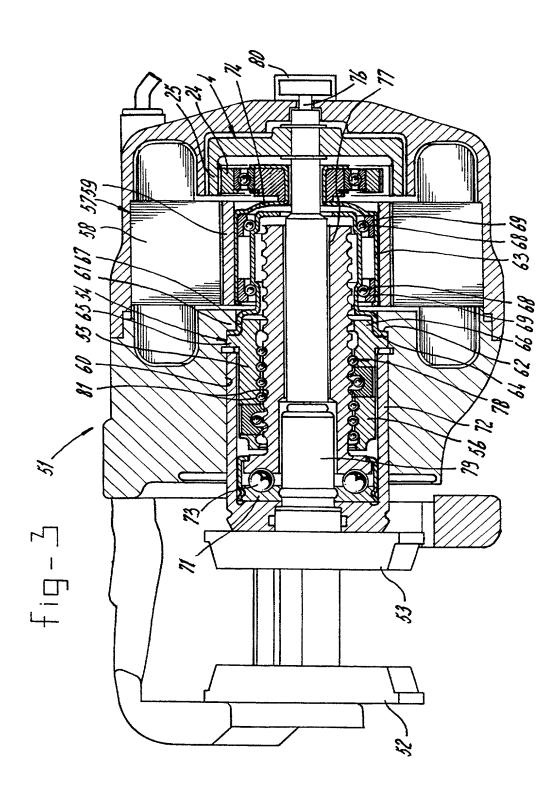
- 16. Actuator according to one claim 1, comprising a housing (55), a mut (55) and a screw (56) one of which is axially fixed with respect to the housing (51) and the other of which is axially displaceable with respect to the housing (51) for moving an actuating head (71), as well as a motor (57) which comprises a stator (58) connected to the housing (51), and a rotor which is drivingly connected to a rotatable part (56) of the screw actuator (54), the housing (51) having a bore (60) accommodating at least the nut (55) and/or screw (56), an axially fixed part (55) of said nut (55) or screw (56) being supported with respect to a radial support abutment (61) which extends inwardly in the bore (60), wherein the rotor (59) of the motor (57) supported rotatably on a sleeve (63), said sleeve (63) engaging the fixed part (55) and extending away from the actuating head (71), said sleeve (63) having a radially outwardly extending sleeve flange (64) which is interposed between said support abutment (61), and the axially fixed part (55).
  - 17. Actuator according to claim 16, wherein the flange (54) of the sleeve (63) is supported on an abutment surface (62) of the support abutment (61) which faces an actuating head (71) connected to the axially displaceable nut (55) or screw (56) for exerting a compressive force.
- 18. Actuator according to claim 16 or 17, wherein the nut (55) is fixedly supported within the housing (51), said nut (55) having a radially out wardly extending nut flange (65) facing the outwardly extending sleeve flange (64) and overlapping the inwardly extending support abutment (61).
  - 19. Actuator according to claim 18, wherein the outwardly facing sturfaces of sleeve flange (14) and the nut flange are curved in axial cross section, so as to allow swivelling or tilting of said nut and sleeve due to misalignment forcess.
  - Actuator according to claim 18 or 19, wherein the nut (55) has a nut extension (66) extending beyond the nut flange (65) and inside the support abutment (61), the sleeve (63) having an axially extending support part (67) which is accommodated between said nut extension (66) and the support abutment (61).

- 31. Actuator according to claim 30, wherein the gear wheel mechanism (4) is connected to a central drive shaft (76) which is rotatably supported in the housing (51) and which extends into a bore (77) in the screw (76) said drive shaft (76) being non-rotatably coupled to the screw (56) through a spline/groove mechanism.
- 32. Actuator according to claim 31, wherein a lubricant dosing module (79) is accommodated in the bore (60) of the screw (66).
- Brake calliper, comprising a claw piece with at least two brakes, and an actuator according to any of the preceding claims.

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d.[] e.[X]	no such applications have been filed
6. [ A ]	such applications have been filed as follow

# EARLIEST FOREIGN APPLICATION(S), IF ANY FILED WITHIN 12 MONTHS (6 MONTHS FOR DESIGN) PRIOR TO SAID APPLICATION

Country	Application Number	Date of filing (day, month, year)	Date of Issue (day, month, year)	Priority claimed
the Netherlands	1011731	6 April 1999		Yes
				·

ALL FOREIGN APPLICATION(S),	IF ANY FILED MORE THAN 12 MONTHS
	PRIOR TO SAID APPLICATION

### CONTINUATION-IN-PART

(Complete this part only if this is a continuation-in-part application)

Thereby declare claim the benefit under Title 35, United States code, paragraph 120 of any United States application(s) listed below and, insofar as the subject matter of each of the claim of this application is not disclosed in the prior United States application in the manner provided by the first paragraph of Title 35, United States Code, paragraph 112, I acknowledge the duty to disclose material information as defined in Title 37, Code of Federal Regulations, paragraph 1.56(a) which occurred between the fifting date of the prior application and the national or PCT international filling date of this application:

(Application Serial No.) (Filing date)	(Status)	(patented, pending, abandoned)
(Application Serial No.) (Filing date)	(Status)	(patented, pending, abandoned)

# **POWER OF ATTORNEY**

Assistant Commissioner for Palante Washington, D.C. 20231

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The undersigned, the above-noted Applicants, hereby revoks all previous powers of attorney and appoint the following as attorneys of record with power of substitution and revocation to prosecute this palent application and all continuations and divisions thereof, and to transact all business in the Patent and Trademark

James A. Oliff, Registration No. 27,076:
Williem P. Berridge, Registration No. 30,024;
Kirk M. Hudson, Registration No. 47,682;
Thomas J. Pardini, Registration No. 30,411; and Edward P. Walker, Registration No. 31,4500.

ALL CORRESPONDENCE IN CONNECTION WITH APPLICATION SHOULD BE SENT TO CLIFF & BERRIDGE, P.O. BOX 18928, ALEXANDRIA, VIRGINIA 22320, TELEPHONE; (703)

# COMBINED DECLARATION AND POWER OF ATTORNEY

(ORIGINAL DESIGN, NATIONAL STAGE OF PCT OR CIP APPLICATION)

As a below named inventor, I hereby declare that

My residence, post office address and citizenship are as stated below next to my name, I believe I am the original, first and sole inventor (if only one name is listed below) or an original, first and joint inventor (if plural names are listed below) of the subject matter which is claimed and for which a patent is sought on the invention entitled:

## Actuator having compact bear reduction

the specification of which: (complete (a), (b) or (c) for type of application)

### REGULAR OR DESIGN APPLICATION

a. [ ] is attached hereto.

b. [ ] was filed on

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, gk

c. [X]

Serial No.

as Application

and was amended on

(if applicable)

# PCT FILED APPLICATION ENTERING NATIONAL STAGE

was described and claimed in International application No. PCT/NL00/00227

filed on 6 April 2000 and as amended on

(if any)

# ACKNOWLEDGEMENT OF REVIEW OF PAPERS AND DUTY OF CANDOR

hereby state that I have reviewed and understand the contents of the above identified specification, including the claims, as amended by any amendment referred to above.

l acknowledge the duty to disclose information which is material to patentability as defined in Title 37, Code of Federal Regulations, paragraph 1.56(a).

In compliance with this duty there is attached an information disclosure statement 37 CFR 1.97

### PRIORITY CLAIM

I hereby claim foreign priority benefits under Title 35. United States Code paragraph 119 of any foreign application (s) for patent of inventor's certificate listed below and have also identified below any foreign application for patent of inventor's certificate having a filing date before that of the application on which priority is claimed.

....... or my own michaed are the and that all statements made on information and belief are believed to be true; and further that these statements were made with the knowledge that wilful false statements and the like so made are punishable by fine or imprisonment, or both under Section 1001 of Title 18 of the United States Code and that such wilful false statements may jeopardize the validity of the application or any patent issued thereon.

Full name of sole or first inventor: KAPAAN, Hendrikus Jan Inventor's signature

8 October 2001

Country of Citizenship: the Netherlands

Residence: Nieuwegein, the Netherland NLX

1 k

Post Office Address: Waterhoen 5, NL-3435 DM NIEUWEGEIN, the Netherlands

Full name of second inventor: ZWARTS, Jacobus

nventor's signature

8 October 2001

Country of Citizenship: the Netherlands

Residence: Nieuwegein, the Netherlands

Post Office Address: Carmenlaan 5, NL-3438 VA NIEUWEGEIN, the Netherland

Full name of third inventor: BROERSEN, Simon Jan

Inventor's signature

8 October 2001

Country of Citizenship: the Netherlands

Residence: Nieuwegein, the Netherlands

Post Office Address: Lichtegaarde 41, NL-3436 ZS NIEUWEGEIN, the Netherlands

CHECK PROPER BOX(ES) FOR ANY ADDED PAGE(S) FORMING A PART OF THIS DECLARATION